



Institute of Aeronautics and Applied Mechanics

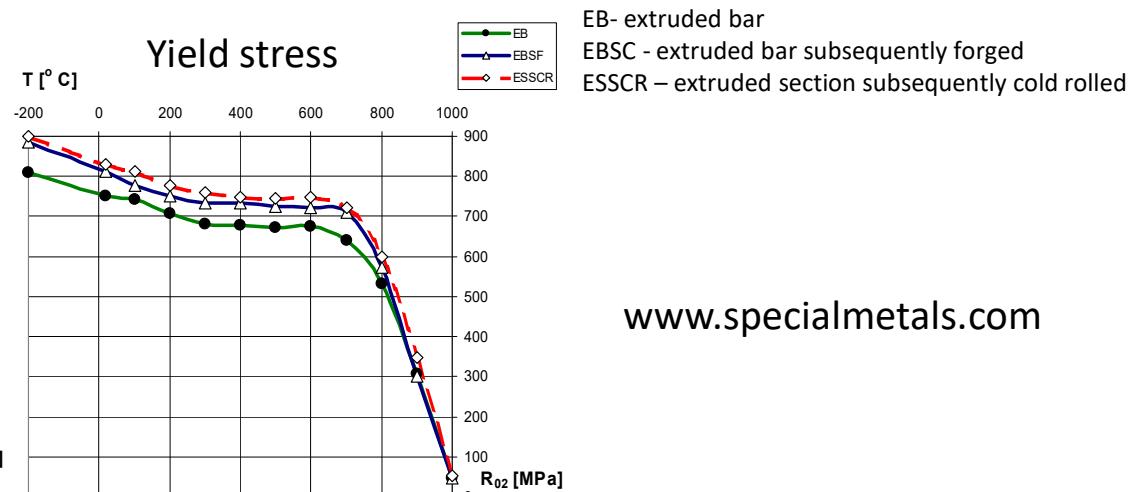
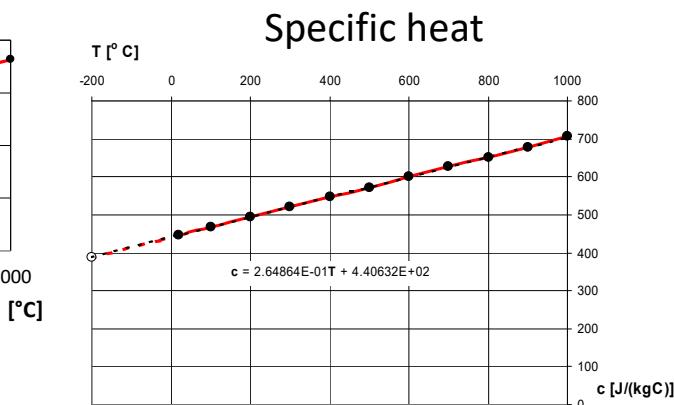
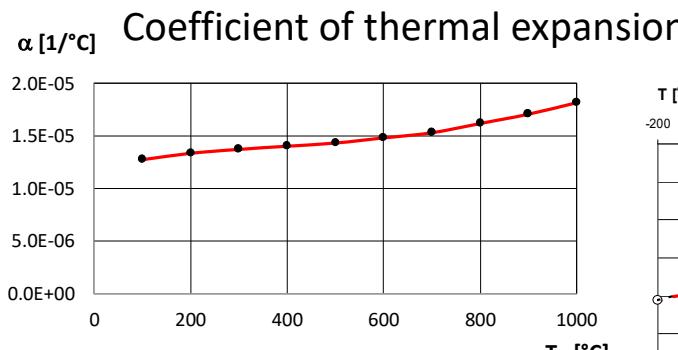
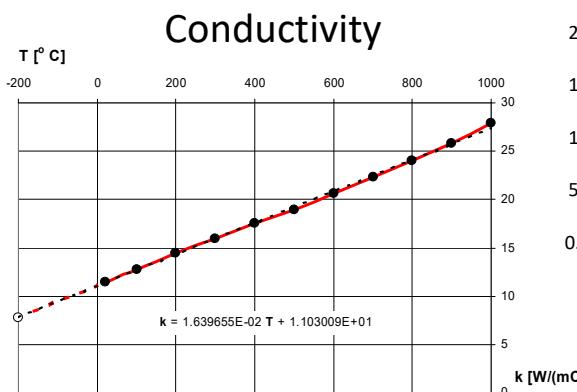
Finite element method 2 (FEM 2)

Heat transfer and thermal stresses

10.2021

Thermo-mechanical properties versus temperature

Nimonic 90: nickel-based high-temperature low creep alloy for use in aircraft and gas turbine components such as turbine blades and engine exhaust jets.



www.specialmetals.com

Secant coefficient of thermal expansion $\alpha_{se}(T)$

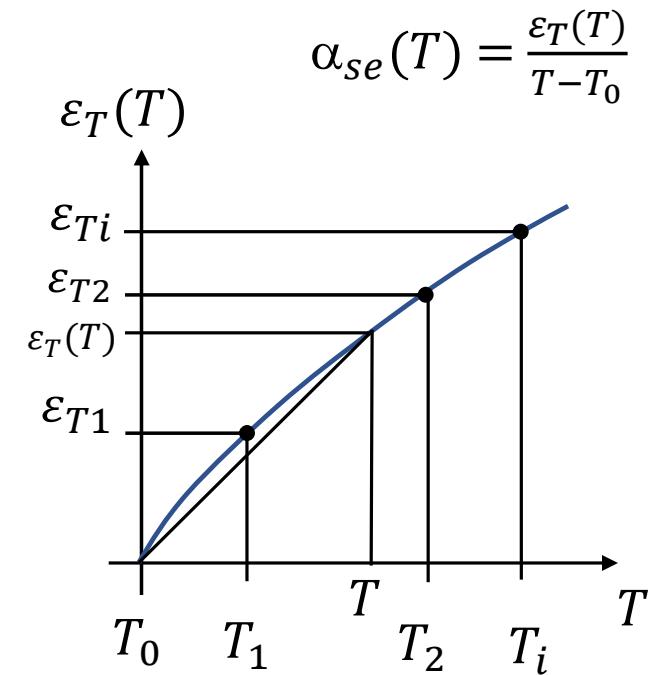
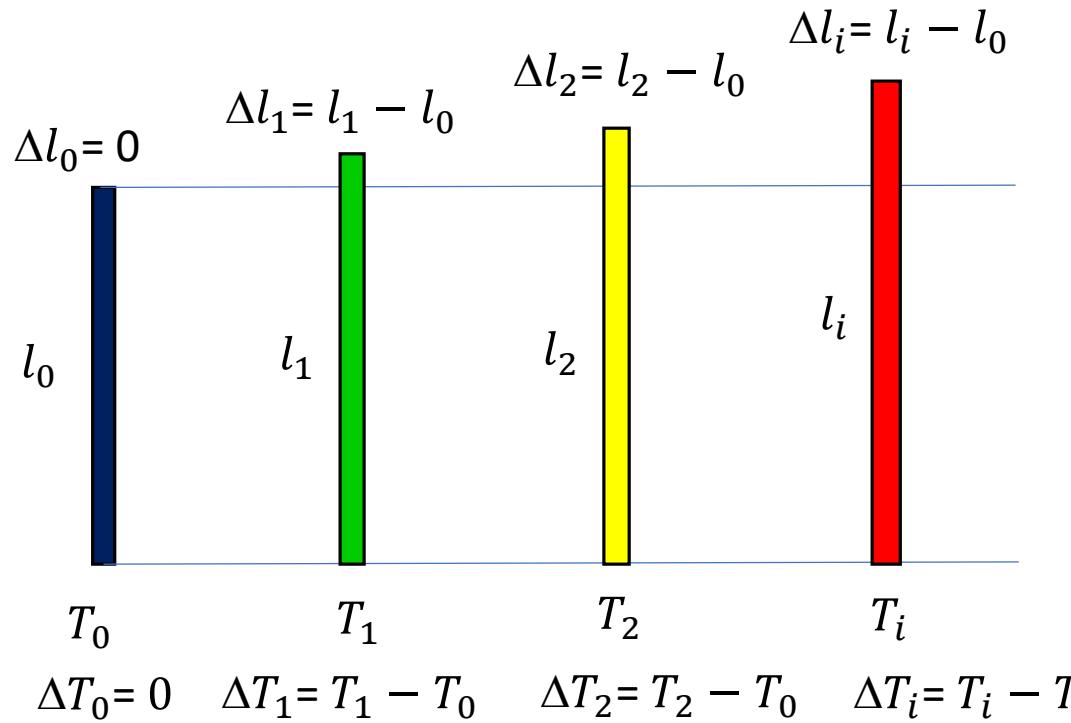
Thermal strain:

$$\varepsilon_T = \frac{\Delta l}{l} = \alpha_{se} \cdot \Delta T$$

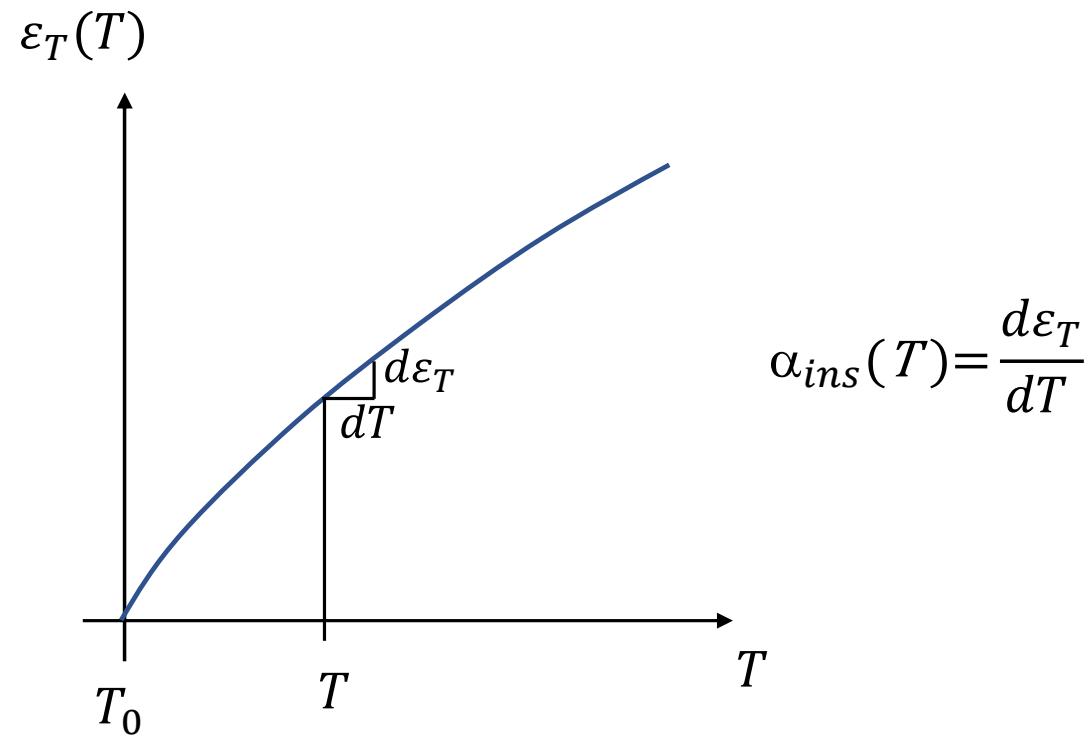
secant coefficient of thermal expansion

$$\varepsilon_{Ti} = \frac{\Delta l_i}{l_0}$$

$$\alpha_{se}(T_i) = \frac{\varepsilon_{Ti}}{\Delta T_i} = \frac{\Delta l_i}{l_0 \Delta T_i}$$



Instantaneous coefficient of thermal expansion $\alpha_{ins}(T)$



Coefficient of thermal expansion

T_0 - temperature at which $\varepsilon_T = 0$ - in the test

T_{REF} - temperature at which $\varepsilon_T = 0$ - for working conditions

for $T_{REF} = T_0$:

$$\alpha_{se}(T) = \frac{1}{(T - T_0)} \int_{T_0}^T \alpha_{ins}(\bar{T}) d\bar{T}$$

$$\varepsilon_T(T) = \alpha_{se}(T) \cdot (T - T_0)$$

if $T_{REF} \neq T_0$, the secant coefficient $\alpha_{se}(T)$ is recalculated

Strain in a structure including the thermal effect

The total strain is a sum of thermal and elastic strain:

$$\{\varepsilon\} = \{\varepsilon\}_T + \{\varepsilon\}_e = \begin{pmatrix} \alpha_x \Delta T \\ \alpha_y \Delta T \\ \alpha_z \Delta T \\ 0 \\ 0 \\ 0 \end{pmatrix}_T + \begin{pmatrix} \varepsilon_x \\ \varepsilon_y \\ \varepsilon_z \\ \gamma_{xy} \\ \gamma_{yz} \\ \gamma_{zx} \end{pmatrix}_e = \begin{pmatrix} \alpha_x \Delta T \\ \alpha_y \Delta T \\ \alpha_z \Delta T \\ 0 \\ 0 \\ 0 \end{pmatrix}_T + [D]^{-1} \begin{pmatrix} \sigma_x \\ \sigma_y \\ \sigma_z \\ \tau_{xy} \\ \tau_{yz} \\ \tau_{zx} \end{pmatrix}$$

For isotropy:

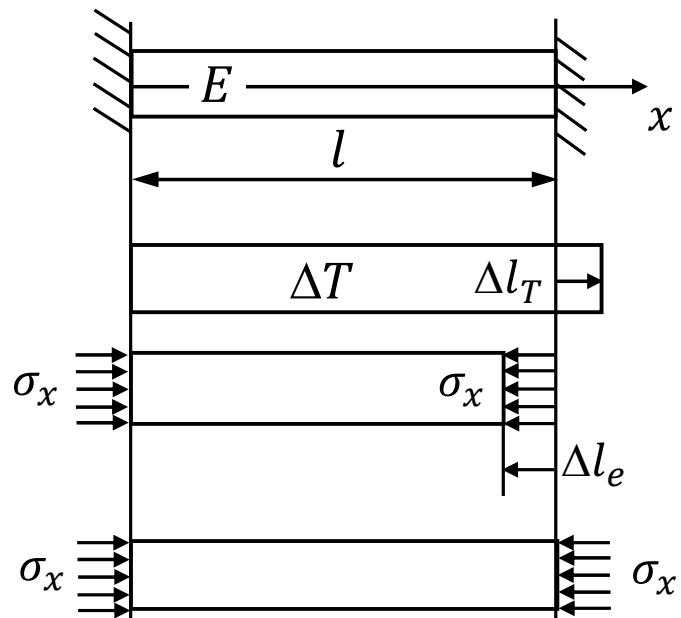
$$\{\varepsilon\} = \alpha_{se} \cdot \Delta T \begin{pmatrix} 1 \\ 1 \\ 1 \\ 0 \\ 0 \\ 0 \end{pmatrix}_T + [D]^{-1} \cdot \{\sigma\}$$

↑
stress vector
↑
inverse constitutive matrix

Thermal stress is observed for:

- nonuniform temperature field
- temperature change and nonhomogeneous material
- temperature change and statically indeterminate constraints

Example: statically indeterminate bar with uniform temperature distribution



$$\Delta l_T + \Delta l_e = 0$$

$$\varepsilon_{xT} \cdot l + \varepsilon_{xe} \cdot l = 0$$

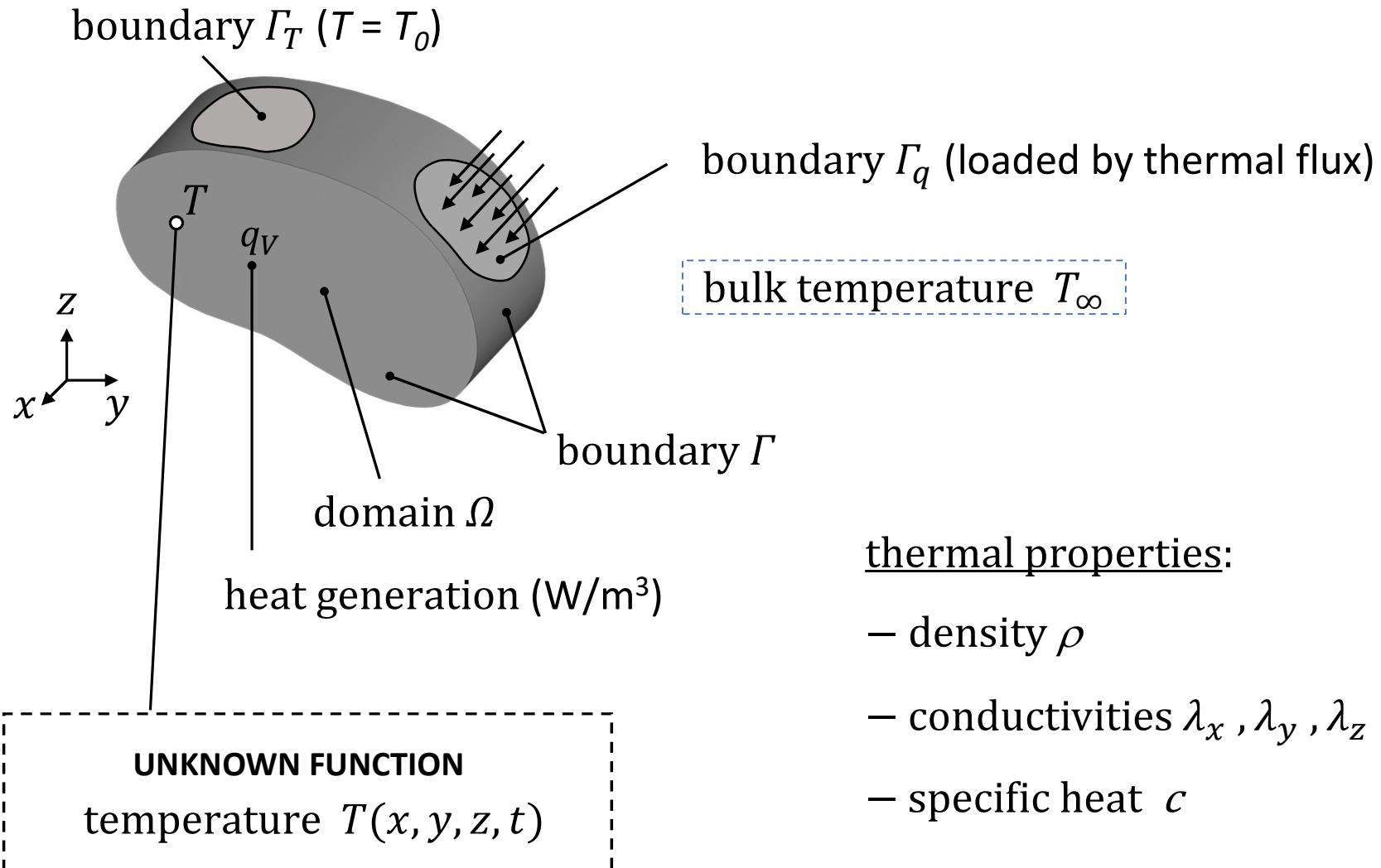
$$\alpha_{se} \cdot \Delta T \cdot l + \frac{\sigma_x}{E} \cdot l = 0$$

$$\sigma_x = -E \cdot \alpha_{se} \cdot \Delta T$$

for: $E = 2 \cdot 10^5 \text{ MPa}$, $\Delta T = 100^\circ\text{C}$, $\alpha_{se} = 1.2 \cdot 10^{-5} \text{ 1/}^\circ\text{C}$:

$$\sigma_x = -240 \text{ MPa} \quad (\text{compression – possible buckling})$$

Boundary value problem of heat transfer



Thermal analysis

Transient heat flow inside a solid body (law of conservation of energy):

$$\frac{\partial}{\partial x} \left(\lambda_x \frac{\partial T}{\partial x} \right) + \frac{\partial}{\partial y} \left(\lambda_y \frac{\partial T}{\partial y} \right) + \frac{\partial}{\partial z} \left(\lambda_z \frac{\partial T}{\partial z} \right) + q_V(x, y, z, t) = \rho c \frac{\partial T}{\partial t}$$

$$\rho c \frac{\partial T}{\partial t} = 0 \quad \text{for a steady state:}$$

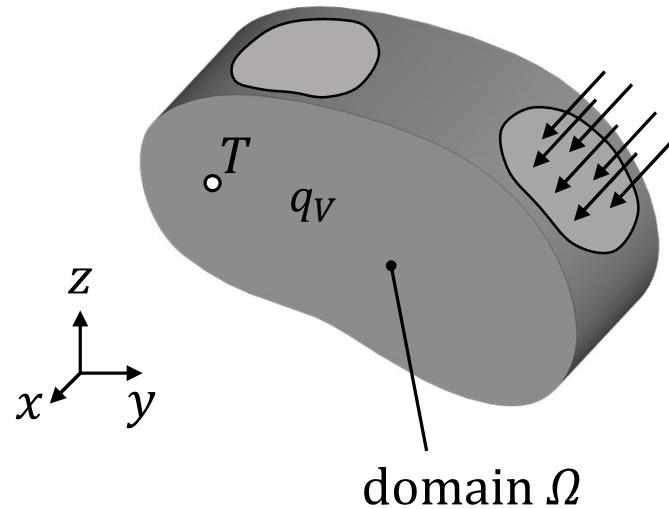
Equivalent functional for steady state analysis :

$$J = \int_{\Omega} \frac{1}{2} \left(\lambda_x \left(\frac{\partial T}{\partial x} \right)^2 + \lambda_y \left(\frac{\partial T}{\partial y} \right)^2 + \lambda_z \left(\frac{\partial T}{\partial z} \right)^2 - 2q_V T \right) d\Omega + \\ + \int_{\Gamma_q} \left(\frac{1}{2} \alpha (T - T_{\infty})^2 + qT \right) d\Gamma_q$$

For isotropy:

$$J = \int_{\Omega} \frac{1}{2} \left(\lambda \left(\left(\frac{\partial T}{\partial x} \right)^2 + \left(\frac{\partial T}{\partial y} \right)^2 + \left(\frac{\partial T}{\partial z} \right)^2 \right) - 2q_V T \right) d\Omega + \int_{\Gamma_q} \left(\frac{1}{2} \alpha (T - T_{\infty})^2 + qT \right) d\Gamma_q$$

Finite element model



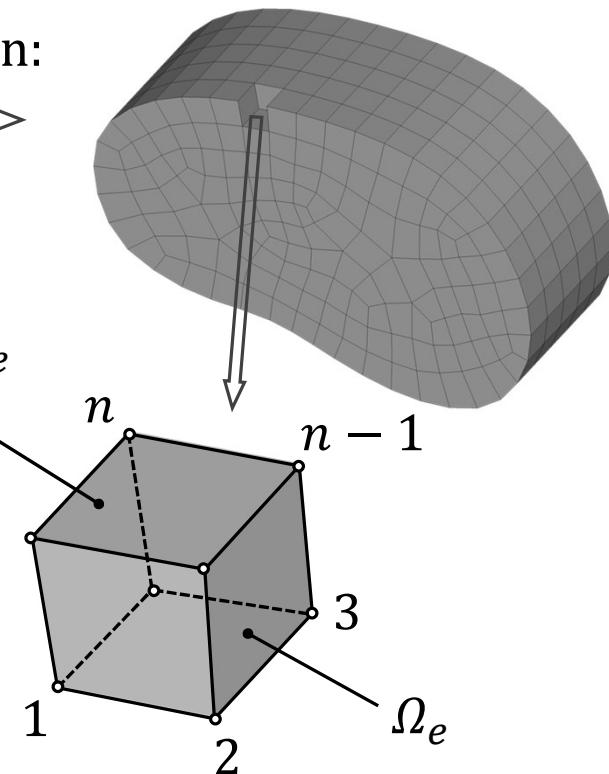
$$\Omega = \sum_{e=1}^{NOE} \Omega_e \text{ and } \Omega_i \cap \Omega_j = 0 \quad i \neq j$$

NOE – no. of finite elements
 NON – no. of nodes

discretization:

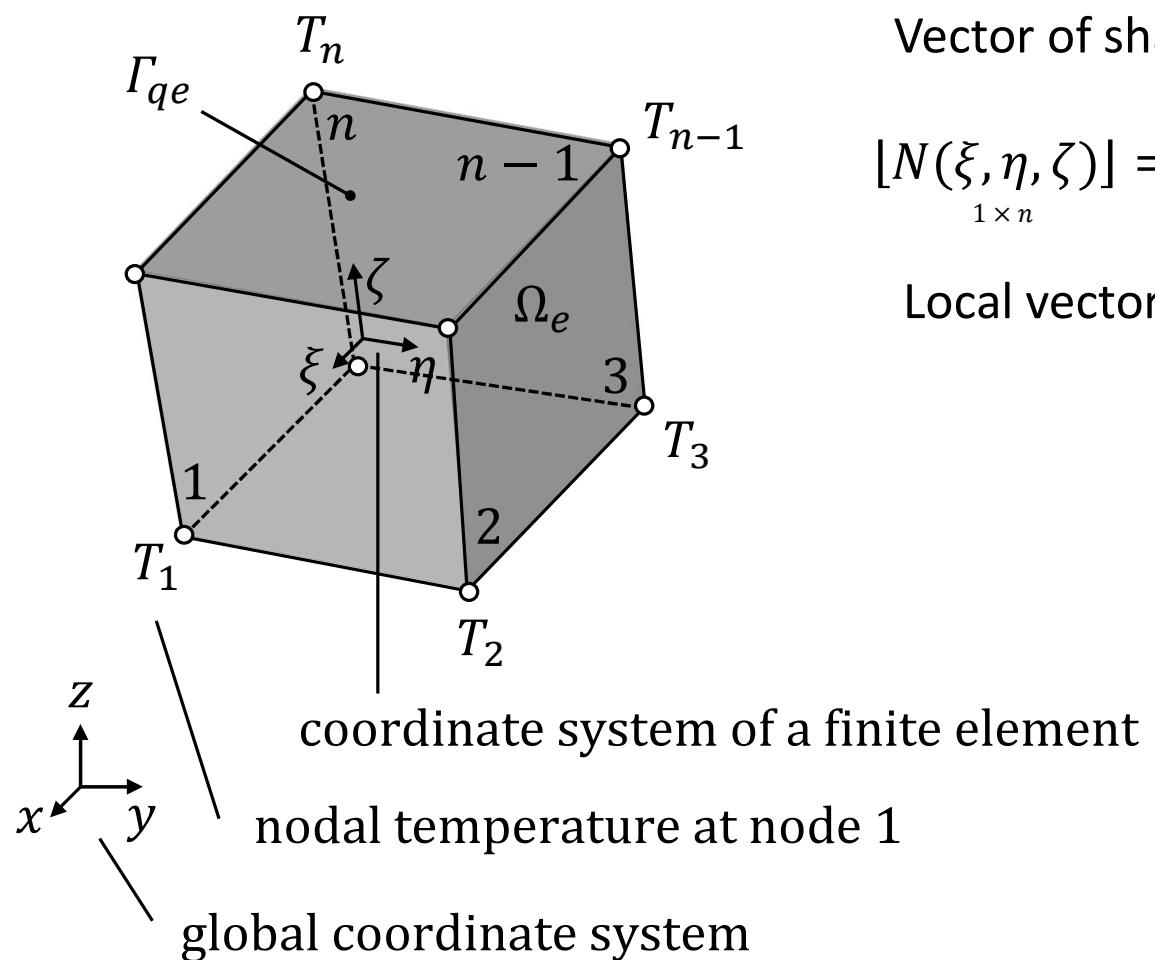


boundary Γ_{qe}



Finite element with n - nodes

Nodal approximation in the finite element for thermal analysis



$$\text{temperature } T(\xi, \eta, \zeta) = [N(\xi, \eta, \zeta)] \{T\}_e$$

$1 \times n$ $n \times 1$

Vector of shape functions

$$[N(\xi, \eta, \zeta)] = [N_1, N_2, \dots, N_i, \dots, N_n]$$

$1 \times n$

Local vector of nodal temperatures

$$\{T\}_e = \left\{ \begin{array}{c} T_1 \\ T_2 \\ T_3 \\ \vdots \\ T_i \\ \vdots \\ T_n \end{array} \right\}_e$$

$n \times 1$

Functional formulation for a finite element (isotropy and steady state):

$$J_e = \int_{\Omega_e} \frac{1}{2} \left(\lambda \left(\left(\frac{\partial T}{\partial x} \right)^2 + \left(\frac{\partial T}{\partial y} \right)^2 + \left(\frac{\partial T}{\partial z} \right)^2 \right) - 2q_V T \right) d\Omega_e + \int_{\Gamma_{qe}} \left(\frac{1}{2} \alpha (T - T_\infty)^2 + q T \right) d\Gamma_{qe}$$

derivative of J_e with respect to temperature T_i :

$$\begin{aligned} \frac{\partial J_e}{\partial T_i} &= \int_{\Omega_e} \left(\lambda \left(\frac{\partial T}{\partial x} \cdot \frac{\partial}{\partial T_i} \left(\frac{\partial T}{\partial x} \right) + \frac{\partial T}{\partial y} \cdot \frac{\partial}{\partial T_i} \left(\frac{\partial T}{\partial y} \right) + \frac{\partial T}{\partial z} \cdot \frac{\partial}{\partial T_i} \left(\frac{\partial T}{\partial z} \right) \right) - q_V \frac{\partial T}{\partial T_i} \right) d\Omega_e \\ &\quad + \int_{\Gamma_{qe}} \left(\alpha (T - T_\infty) \frac{\partial T}{\partial T_i} + q \frac{\partial T}{\partial T_i} \right) d\Gamma_{qe} \end{aligned}$$

$$T(\xi, \eta, \zeta) = [N(\xi, \eta, \zeta)] \{T\}_e = [N_1, N_2, \dots, N_i, \dots, N_n] \begin{Bmatrix} T_1 \\ T_2 \\ T_3 \\ \vdots \\ T_i \\ \vdots \\ T_n \end{Bmatrix}_e$$

$$1 \times n \qquad n \times 1$$

$$\frac{\partial T}{\partial T_i} = N_i$$

$$\frac{\partial T}{\partial x} = \frac{\partial [N]}{\partial x} \{T\}_e + [N] \frac{\partial \{T\}_e}{\partial x} = \left[\frac{\partial N_1}{\partial x}, \frac{\partial N_2}{\partial x}, \dots, \frac{\partial N_i}{\partial x}, \dots, \frac{\partial N_n}{\partial x} \right] \begin{Bmatrix} T_1 \\ T_2 \\ T_3 \\ \vdots \\ T_i \\ \vdots \\ T_n \end{Bmatrix}_e$$

\uparrow
 $\frac{\partial \{T\}_e}{\partial x} = \{0\}$

$$\frac{\partial N_i}{\partial x} = \frac{\partial N_i}{\partial \xi} \cdot \frac{\partial \xi}{\partial x} + \frac{\partial N_i}{\partial \eta} \cdot \frac{\partial \eta}{\partial x} + \frac{\partial N_i}{\partial \zeta} \cdot \frac{\partial \zeta}{\partial x} \quad (\text{Jacobian matrix})$$

$$\frac{\partial}{\partial T_i} \left(\frac{\partial T}{\partial x} \right) = \frac{\partial N_i}{\partial x}$$

similarly:

$$\frac{\partial T}{\partial y} = \left[\frac{\partial N_1}{\partial y}, \frac{\partial N_2}{\partial y}, \dots, \frac{\partial N_i}{\partial y}, \dots, \frac{\partial N_n}{\partial y} \right] \begin{Bmatrix} T_1 \\ T_2 \\ T_3 \\ \vdots \\ T_i \\ \vdots \\ T_n \end{Bmatrix}_e$$

$$\frac{\partial T}{\partial z} = \left[\frac{\partial N_1}{\partial z}, \frac{\partial N_2}{\partial z}, \dots, \frac{\partial N_i}{\partial z}, \dots, \frac{\partial N_n}{\partial z} \right] \begin{Bmatrix} T_1 \\ T_2 \\ T_3 \\ \vdots \\ T_i \\ \vdots \\ T_n \end{Bmatrix}_e$$

$$\frac{\partial N_i}{\partial y} = \frac{\partial N_i}{\partial \xi} \cdot \frac{\partial \xi}{\partial y} + \frac{\partial N_i}{\partial \eta} \cdot \frac{\partial \eta}{\partial y} + \frac{\partial N_i}{\partial \zeta} \cdot \frac{\partial \zeta}{\partial y}$$

$$\frac{\partial N_i}{\partial z} = \frac{\partial N_i}{\partial \xi} \cdot \frac{\partial \xi}{\partial z} + \frac{\partial N_i}{\partial \eta} \cdot \frac{\partial \eta}{\partial z} + \frac{\partial N_i}{\partial \zeta} \cdot \frac{\partial \zeta}{\partial z}$$

$$\frac{\partial}{\partial T_i} \left(\frac{\partial T}{\partial y} \right) = \frac{\partial N_i}{\partial y}$$

$$\frac{\partial}{\partial T_i} \left(\frac{\partial T}{\partial z} \right) = \frac{\partial N_i}{\partial z}$$

derivative of J_e with respect to temperature T_i :

$$\begin{aligned} \frac{\partial J_e}{\partial T_i} = & \int_{\Omega_e} \lambda \left(\underbrace{\frac{\partial [N]}{\partial x} \cdot \frac{\partial N_i}{\partial x}}_{1 \times n} + \underbrace{\frac{\partial [N]}{\partial y} \cdot \frac{\partial N_i}{\partial y}}_{1 \times n} + \underbrace{\frac{\partial [N]}{\partial z} \cdot \frac{\partial N_i}{\partial z}}_{1 \times n} \right) d\Omega_e - \int_{\Omega_e} q_V N_i d\Omega_e \\ & + \int_{\Gamma_{qe}} (\alpha([N]\{T\}_e - T_\infty) N_i + q N_i) d\Gamma_{qe} \end{aligned}$$

derivatives of J_e with respect to temperatures $T_1, T_2, \dots, T_i, \dots, T_n$:

$$\frac{\partial J_e}{\partial T_1}, \frac{\partial J_e}{\partial T_2}, \dots, \frac{\partial J_e}{\partial T_i}, \dots, \frac{\partial J_e}{\partial T_n}$$



$$\frac{\partial J_e}{\partial \{T\}_e} = \underbrace{[h]_e}_{n \times 1} \underbrace{\{T\}_e}_{n \times n} + \underbrace{\{F\}_e}_{n \times 1}$$

local conductivity matrix

thermal load vector in a finite element

components of the local conductivity matrix:

$$h_{ij} = \int_{\Omega_e} \lambda \left(\frac{\partial N_i}{\partial x} \cdot \frac{\partial N_j}{\partial x} + \frac{\partial N_i}{\partial y} \cdot \frac{\partial N_j}{\partial y} + \frac{\partial N_i}{\partial z} \cdot \frac{\partial N_j}{\partial z} \right) d\Omega_e$$

components of the thermal load vector:

$$F_{ie} = - \int_{\Omega_e} q_V N_i d\Omega_e + \int_{\Gamma_{qe}} \alpha ([N] \{T\}_e - T_\infty) N_i d\Gamma_{qe} + \int_{\Gamma_{qe}} q N_i d\Gamma_{qe}$$



heat generation



convection & radiation



applied thermal flux

3 ways of heat transfer

- conduction

$$q = -\lambda \cdot \text{grad}(T)$$

rate of heat flow per unit area (HEAT FLUX)

- convection

$$q = \alpha(T - T_\infty)$$

- radiation

$$\alpha = f(T) = c \frac{T^4 - T_{\infty}^4}{T - T_{\infty}}$$

Medium (fluid)	Free convection	Forced convection
gas (air)	5–30	30–500
water	30–300	300–20000
oil	5–100	30–3000
liquid metals	50–500	500–20000

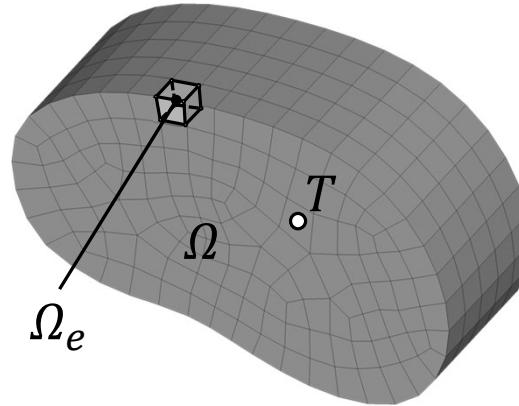
$$\left(\frac{W}{m^2 K} \right)$$

Minimalization of equivalent functional J for the entire FE model:

NOE – no. of FEs

NON – no. of nodes

$NON=NDOF$



To find the solution of heat transfer, the derivative of functional J with respect to the global vector of nodal temperatures is minimized:

$$\frac{\partial J}{\partial \{T\}} = [H] \cdot \{T\} + \{F\} = \{0\}$$

global conductivity matrix

global vector of nodal temperatures

global thermal load vector

Including boundary conditions:

$$N = NO_N - NO_F$$

number of known nodal temperatures

Set of algebraic equation after taking boundary conditions into account:

$$\begin{matrix} [H] \cdot \{T\} + \{F\} = \{0\} \\ N \times N \quad N \times 1 \quad N \times 1 \quad N \times 1 \end{matrix}$$